

| | 2.1 . Coue. 17. | <u>vil</u> 203 | 11 | | | | | | | | | Ľ | | |
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| F | Reg. No: | | | | | | | | | | | | | |
| | SIDDH | | H INS | STITU | ТЕ О | FEN | GINE | ERIN | IG & ' | ГЕСН | INOI | | PUTTU | R |
| | | | | , | 12 0 | (AU | TON | OMOL | JS) | 1201 | | | | |
| | B.1 | ech l | II Yea | ar I Se | emest | ter Su | apple | ment | ary E | xami | natio | ns July | -2022 | |
| | | | | DES | SIGN | OF M | IACH | INE F | ELEM | ENT | S-I | | | |
| | | | | | (N | Aecha | nical l | Engine | ering |) | | | | |
| Т | ime: 3 hours | | | | | | | | | | | | Max. N | Aarks: 60 |
| | | | | (Ans | swer a | ll Five | e Units | s 5 x 1 | 2 = 6 | 0 Mar | ks) | | | |
| | | | | | | | UNI | T-I | | | | | | |
| 1 | a How do y | vou cla | ssify | materi | als for | engir | neering | g use? | | | | | L1 | 6 M |
| | b What are | the ge | eneral | design | o consi | iderati | on she | ould b | e follo | wed w | while | designin | g L1 | 6M |
| | amachine | eleme | ent | | | | | | | | | | | |
| | | | | | | | 0 | R | | | | | | |
| 2 | The load on | a bolt | cons | ists of | an ax | ial pu | ll of 1 | l0 kN | togeth | ner wi | th a t | ransvers | e L2 | 12 M |
| shear force of 5 kN. Find the diameter of bolt required according to 1. Maximum | | | | | | n | | | | | | | | |
| | principal str | ess the | eory; | 2.Max | imum | shea | r stres | s theo | ory; 3 | .Maxi | mum | principa | ıl | |
| | strain theory | ,4.Maz | x1mur | n stran | n ener | gy the | ory; a | nd 5.N | /lax1m | um di | stortic | on energy | У | |
| | theory. | | | | | | | | | | | | | |
| _ | | | | | | | UNI | <u>[-]]</u> | | _ | | _ | | |
| 3 | a What are Stress – T | the flu | ictuat | ting str dal cur | ess, re | epeate | d stres | ss and | rever | sed st | ress? | Draw the | e L3 | 6M |
| | b Determine | e the | diame | eter of | a cir | cular | rod n | nade d | of duc | tile n | nateria | al with | a L2 | 6M |
| | fatigue str | ength | (com | plete r | eversa | l), σe | =265 1 | MPa a | nd ten | sile vi | ield st | rength o | f | 0112 |
| | 350 MPa. | The n | nemb | er is su | bjecte | ed to a | ı varyi | ng axi | al loa | d fron | n W n | $\sin = -30$ | 0 | |
| | KN to W | max | = 70 | 0 KN | and h | as a s | stress | conce | ntratic | n fact | tor is | 1.8. Us | e | |
| | factor of s | afety | as 2. | | | | | | | | | | | |
| | | | | | | | 0 | R | | | | | | |
| 4 | A machine c | ompor | nent i | s subje | ected t | o a fl | exural | stress | whic | h fluc | tuates | between | 1 L1 | 12 M |
| | + 300 MN/r | n2 and | 1 – 1 | 50 MN | V/m2. | Deter | mine | the va | alue o | f mini | imum | ultimate | 9 | |
| | strength acco | ording | to 1. | Gerbe | r rela | tion; 2 | 2. Mo | dified | Good | man r | elatio | n; and 3 | • | |
| | Soderberg re | elation | . Tak | te yiel | d stre | ngth | = 0.55 | 5 Ulti | mate | streng | th; E | ndurance | 3 | |
| | strength $= 0$. | 5 Ultır | nate s | strengtl | n; and | actor | of safe | ety = 2 | 2. | | | | | |
| | | | | | | | UNI | '-III | | | | | | |
| 5 | a Two macl | ine pa | arts ar | e faste | ned to | gethe | r tight | ly by 1 | neans | of a 2 | 4 mm | tap bolt | i. L2 | 6M |
| | If the load | l tendi | ng to | separa | te thes | se part | s is ne | eglecte | ed, fino | the s | tress | that is se | я Я | |
| | up in the l | ouded | une 1 | mual ti | ignten | ing. പ | motor | of 1 | 00 | n ond | l tha | blow of | τo | <i>C</i> M |
| | J A lever I | oaued | sale | iy valv | e nas | a ula | ameter | UI I | oo mi | iii alio | i ine | UIUW OI | | UIVI |

pressure is1.6 N/mm2. The fulcrum of the lever is screwed into the cast iron body of the cover. Find the diameter of the threaded part of the fulcrum if the permissible tensile stress is limited to 50 MPa and the leverage ratio is 8.

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Q.P. Code: 19ME0311

OR

6 Fig. shows a solid forged bracket to carry a vertical load of 13.5 kN applied L3 12 M through the centre of hole. The square flange is secured to the flat side of a vertical stanchion through four bolts. Calculate suitable diameter D and d for the arms of the bracket, if the permissible stresses are 110 MPa in tension and 65 MPa in shear. Estimate also the tensile load on each top bolt and the maximum shearing force on each bolt.



UNIT-IV

| 7 | a What are the applications of a cottered joint? | L1 | 6M |
|----|---|----|------|
| | b A knuckle joint is required to withstand a tensile load of 25 kN. Design the | L6 | 6M |
| | joint if the permissible stresses are : $\sigma t= 56$ MPa ; $\tau = 40$ MPa and $\sigma c= 70$ | | |
| | MPa. | | |
| | OR | | |
| 8 | Design a sleeve and cotter joint to resist a tensile load of 60 kN. All parts of the | L6 | 12 M |
| | joint are made of the same material with the following allowable stresses: Tensile | | |
| | stress = 60 MPa; shear stress = 70 MPa; and compressive stress = 125 MPa. | | |
| | UNIT-V | | |
| 9 | a What is a key? State its function with neat sketch. | L1 | 6M |
| | b Design the rectangular key for a shaft of 50 mm diameter. The shearing and | L6 | 6M |
| | crushing stresses for the key material are 42 MPa and 70 MPa. | | |
| | OR | | |
| 10 | a Discuss the function of a coupling. Give at least three practical applications. | L2 | 6M |
| | b Design and make a neat dimensioned sketch of a muff coupling which is used | L6 | 6M |
| | to connect two steel shafts transmitting 40 kW at 350 r.p.m. The material for | | |
| | the shafts and key is plain carbon steel for which allowable shear and crushing | | |
| | stresses may be taken as 40 MPa and 80 MPa respectively. The material for | | |
| | the muff is cast iron for which the allowable shear stress may be assumed as | | |
| | 15 MPa | | |
| | | | |